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Establishment of correlations for the thermo-hydraulic parameters due to perforation in a multi-V rib roughened single pass solar air heater

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ABSTRACT

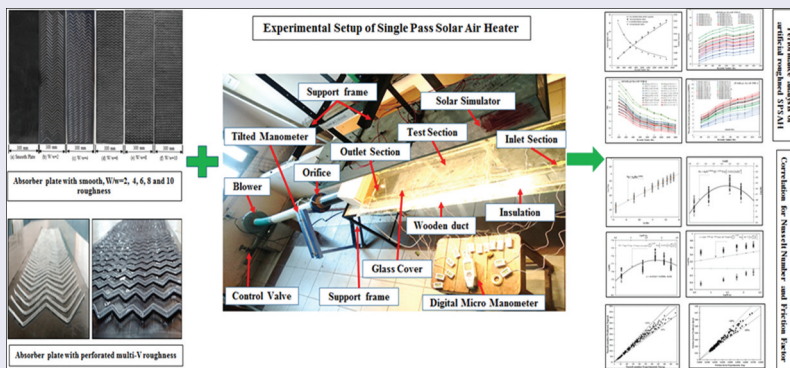
The effect of variation in open area ratio (β) in a multi-V rib roughened Single-Pass Solar Air Heater (SPSAH) is analyzed in this work. Fixing values of other parameters, such as open area ratio (β) and relative roughness width (W/w), were studied with a certain range of Reynolds number (Re) ranging from 2000 to 18000. The improvement in Nusselt number ratio (Nu/Nus) and thermo-hydraulic performance (THPP) and the decline in duct friction ratio (f/fs) support the application of perforation. The empirical correlations of Nu and f are also generated with $\pm 12\%$ and $\pm 7.5\%$ accuracy.

ARTICLE HISTORY

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KEYWORDS

Solar energy; solar air heater; correlation; open area ratio; multi-V rib



Introduction

Solar, wind, hydro, tidal, geothermal, and biomass energy are now the most common renewable energy sources [1–3]. Due to large availability and high level of acceptability solar energy having larger applications in domestic and commercial uses either in thermal and photovoltaic conversion [4]. A Solar Air Heater [SAH] is a basic, low-cost, corrosion-free equipment that easily gathers thermal energy from the sun and conveys it to a working medium [5]. At low-temperature gradients, SAHs have been used in commercial ventilation and air conditioning, process heat, agricultural drying, greenhouse dryers, air-water and space heating, distillation, and textile industries [6]. One of the main causes of SAH's low heat transmission is the lower rate of convective coefficient of heat transfer through the air and the roughened absorber plate.

Many studies have given a variety of design configurations with various roughness forms and sizes that have been compared to the smooth kind of SAH to improve SAH Thermo-Hydraulic Performance (THP) [7–9]. To obtain an optimal combination of roughness forms, several experimental investigations [10, 11], numerical simulation and ANN studies [12–14], CFD analysis [15], and thermodynamics correlations [16, 17] incorporating synthetic roughness of various types, shapes, patterns, widths, orientations, rib thickness, and continuity toward flow were conducted [18]. Multi-V-ribs with different forms of design parameters configurations like size, shapes, continuity, staggering, and orientation were studied by different research scholars [19, 20] and observed significant improvement in thermos-hydraulic performance.

To study perforation impact on flow characteristics, various studies have been conducted on perforation like for shape-full and half perforation [21], circularity of hole [22, 23], and percentage of open area ratio (β) [24], relative hole location [25]. The open area ratio (β) [26–28] and recirculating frequency [29, 30] were shown to have a substantial influence on Nusselt number fluctuation. Varun et al. [27, 31] studied the effect of perforation in multi-v rib roughened SAH and observed significant improvement in Nu/Nu_s and THP while keeping a low friction factor.

To obtain an optimal combination of roughness forms and parameters, thermodynamic correlations for SAH were developed by various research scholars. Saini and Saini [32] developed correlations for expanded metal mesh roughened ducts, while Varshney and Saini [33] developed correlations and conducted numerical studies for SAH packed with wire mesh. Both observed significant improvement in THP. Singh et al. [34] established correlations for Nu and f for discrete V-down ribs of roughness installed in SAH. The optimum values of Nu and f were observed to be 3.04 and 3.11 times higher as compared to smooth SAH, respectively. Hans et al. [35] developed correlations for SAH having multiple v-rib roughnesses, the highest values of Nu and f being determined as 6 and 5 times higher as compared to the smooth SAH. The optimum THP was obtained at $W/w = 6$, whereas the highest f observed at $W/w = 10$ at $\alpha = 60^\circ$. Saini and Verma [36] derived correlations for THP for SAH having dimple-shape roughness and improvement in Nu and f values observed as 7.58 and 4.68 times as compared to the smooth channel. Saini and Saini [37] conducted experimental and numerical studies on SAH and developed correlations for Nu and f for SAH with arc-shaped wire roughness and found an improvement of 3.8 times and 1.75 times as compared to smooth SAH. Sethi et al. [38] developed correlations for artificially roughened SAH with dimpled shape roughness and significant improvement in THP. Chamoli and Thakur [39] derived correlations for SAH with perforated baffles having V-shape roughness, while Kumar et al. [40] developed correlations of THP characteristics for 3-sided roughened SAH and Hans et al. [41] derived correlation function for SAH roughened with broken-arc ribs. All three observed improved THP as compared to smooth SAH.

The open area ratio (β) and the recirculation period have both shown a substantial effect on SAH's effectiveness [31]. Varun et al. [26] studied the influence of change in open area ratio (β) and relative roughness width (W/w) in SPSAH with different perforated multi-V rib and observed substantial improvement in performance.

According to a review, perforation in multi-V rib roughness enhances the THP of SAH due to secondary flow developed by holes, resulting in increased turbulence during separation and reattachment [42]. Only a few studies are available with correlation development on impact analysis of variation in β on THP of SAH. Keeping this in mind, the current research further extends Varun et al. [27] work done to analyses the effect of varying β and W/w values and of perforated multi-V rib roughness on THP for the development of correlations for Nu and f as a function of variable parameters of perforated multi-V roughened SPSH.

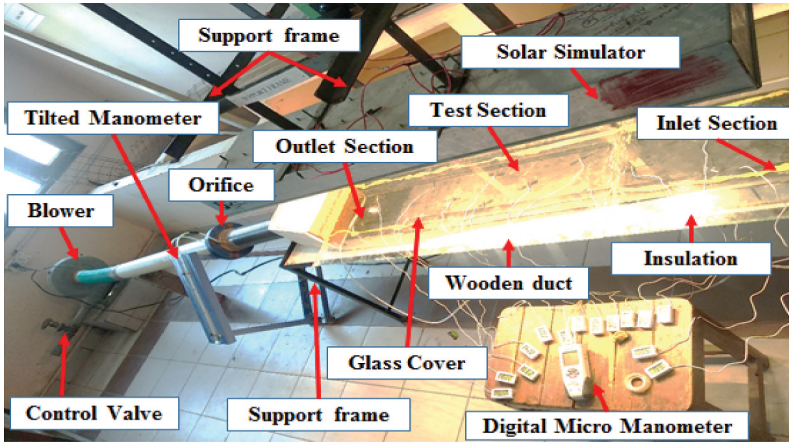


Figure 1. Experimental setup of SPSAH setup.

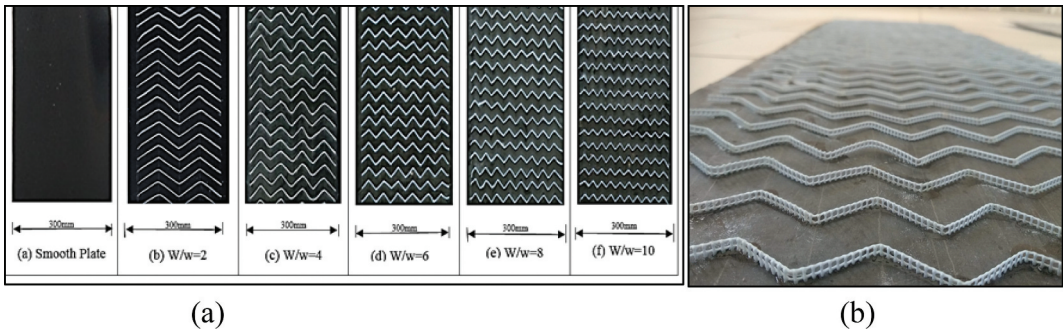


Figure 2. (a) Test plate with smooth, $W/w = 2, 4, 6, 8,$ and 10 perforated multi-V rib roughness (b) Perforated-V shape ribs roughness

Setup details

Designed setup configuration and roughness parameters

The ASHRAE standard (ASHRAE 93-77, 1977) was used to create a SAH configuration with perforated multi-V rib roughness as shown in Figure 1. The details of setup configuration and roughness parameters are discussed in a previous study conducted by Varun et al. [27].

The perforated multi-V rib configurations explored in this work are shown in Figure 2. The configuration, range, and values of ribs variables used in the experimental study were derived from existing literature and mentioned in Table 1.

The parameter open area ratio (β) is a ratio of the area of total perforation in the rib and the rib frontal area and can be defined as:

$$\beta = n \left(\pi * \frac{d_h^2}{4} \right) / b * e \quad (1)$$

Where n = number of holes in one rib.

Table 1. Details of parameters used in the experimental investigation.

Parameter with Unit	Value
Width of Duct (W), m	0.3
Elevation of Duct both side (H), m	0.025
Duct aspect ratio (W/H)	12
Relative roughness height (e/D)	0.043
Relative roughness pitch (p/e)	10
The angle of attack (α), in degree	60
Collector slope (β), in degree	0
Relative roughness width (W/w)	2: 4: 6: 8: 10 (five vales)
Open area ratio (β)	0.0(Continuous rib), 0.21, 0.27, 0.31 (four values)
Reynolds Number (Re)	2000:18000 (nine values)
Ribs Type	Multi-V Ribs
Average insolation (I), W/m ²	800

Data handling and setup validation

Parameters such as pressure gradient and fluid temperature were gathered for the quasi-state, and the hot air mass flow-rate was estimated using the average airflow temperature and channel pressure difference to explore the impacts of various variables on Nu, *f*, and THPP. The detailed description of equations associated with data handling, set-up validation and uncertainty analysis were discussed in a previous study conducted by Varun et al. [27].

An experimental set-up of SAH is being used to conduct a validation study, which includes a traditional smooth duct with a Re ranging from 2000 to 18000. Testing findings for Nu and *f* compared to modified Dittus–Boelter and modified Blasius equations [35], correspondingly as shown in Figure 3.

By using modified Dittus–Boelter equation:

$$Nu_s = 0.024Re^{0.8}Pr^{0.4} \tag{2}$$

By using modified Blasius equation:

$$f_s = 0.085Re^{-0.025} \tag{3}$$

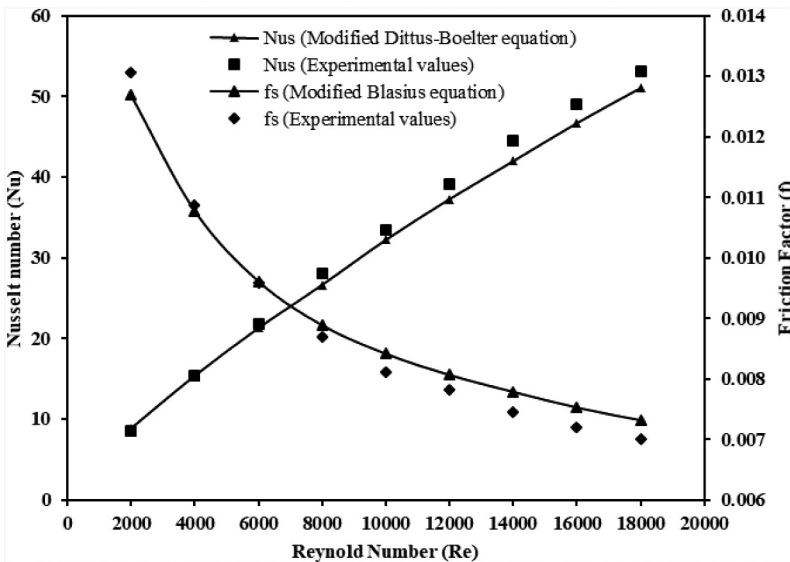


Figure 3. Nus and fs measured and anticipated results for a smooth SAH.

Nu , Re , and f possess maximum percentage errors of $\pm 1.83\%$, $\pm 1.65\%$, and $\pm 3.28\%$, correspondingly.

Results and discussion

The influence of operational and rib variables on flow distribution, THP and mass flow rate is discussed and compared among flat channels for perforated multi-v ribs roughened SPSAH.

Influence of operating parameter

Influence of Reynolds number (Re)

Figure 4(a) shows a comparison of Nu in all cases, indicating a significant gain in Nu at higher values of Re due to a significant reduction in secondary flow and improved mixing and turbulence due to perforation, resulting in a higher Nu and increased thermal performance. The value of Nu observed as 439.80 at $W/w = 6$ for $\beta = 0.27$, at $Re = 18000$, respectively. Figure 4(b) illustrates a considerable reduction in f attributable to secondary flow created by perforation in multi-V ribs, which lowers fluid stream resistance and saves energy during air circulation. The lowest value of f is 0.0128 for perforated Multi-V Ribs with $\beta = 0.31$ at $W/w = 2$ and $Re = 18000$. Perforation in Multi-V ribs decreases considerable friction and power usage under the same operating condition.

Influence of relative roughness width (W/w)

Figure 5 shows an initial increase in W/w enhances Nu in all four conditions for the number of variables examined, with the optimal value at $W/w = 6$. Further increases in the W/w value lower the Nusselt number further. Where as Figure 6 shows that at $W/w = 6$, fluid mixing caused by roughness attained its optimal values, and a further increase in W/w could obstruct secondary flow formation, leading to a decrease in Nu ; the f values continue to rise due to the turbulence formed at high W/w values.

Influence of open area ratio (β)

Figure 7(a) illustrates the influence of β on Nu . This figure shows that Nu is rising as β rises to 0.27, then decreases as a result of the further increase in β value. It can be explained that as the hole diameter gets bigger, the radial development of secondary flow will be stronger for a given axial distance. As an outcome, more fluid mixing takes place in the secondary flow zone, which helps to minimize the flow zone formation on the backside of each rib. It is apparent from this explanation that the β value should be regulated in such a manner that it may enhance the blending of the flow going through the hole to generate local turbulence for enhanced heat transfer among the fluid and the absorber plate. Figure 7 (b) depicts the impact of a change in β value on the f while keeping other parameters same. Because a greater value of β offers less resistance to flow, the f value decreases as the β value increases, resulting in a low f value.

Influence on Nusselt Number ratio (Nu/Nus) and friction factor ratio (f/fs)

Figure 8(a) represents the relationship among Nu/Nus with Re for different perforated multi-V roughness. For all W/w values, the perforated rib with $\beta = 0.27$ performs optimally. Further increases in β values lower Nu/Nus , which is reasonable given that fluid mixing induced by roughness achieved its optimal value at $\beta = 0.27$ and that a further rise in values might hinder the formation of secondary flow, resulting in a reduction in Nu/Nus values.

The effect of Re on f/fs for different β values is shown in Figure 8(b). The trend demonstrates that when β increases f/fs falls due to the formation of strong secondary flow via the hole.

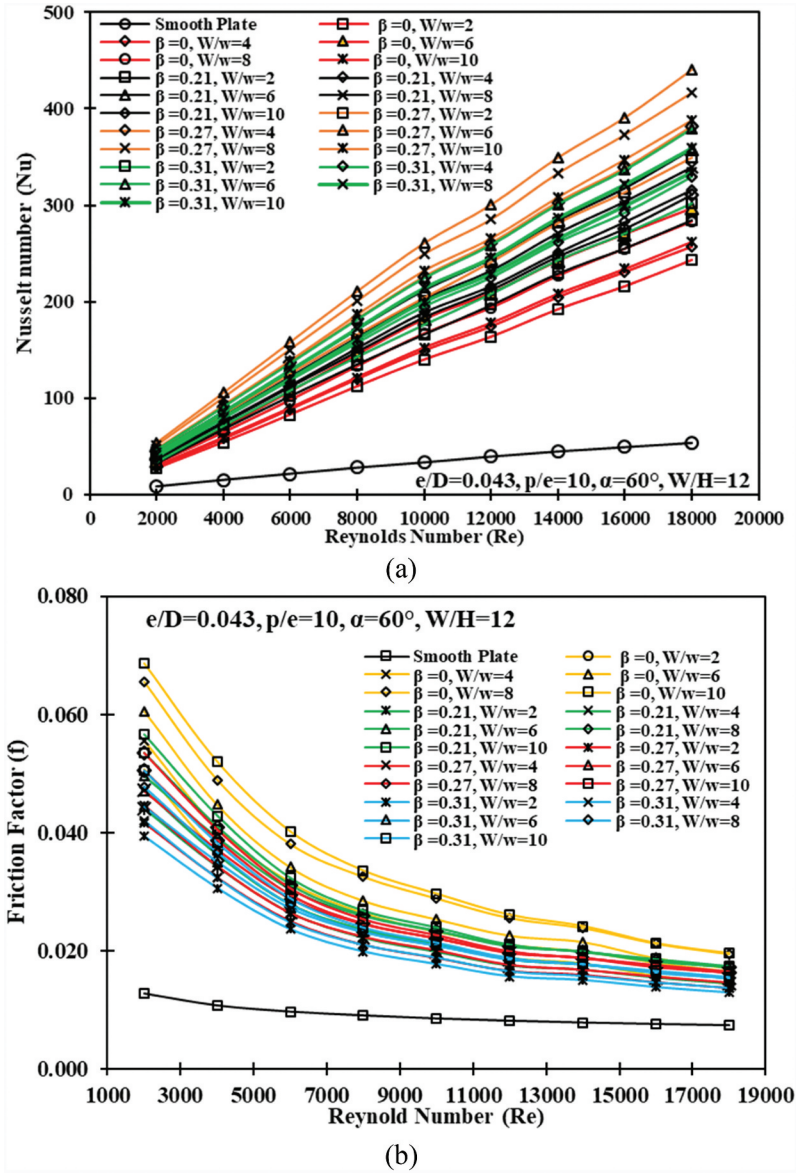


Figure 4. Influence of Re on (a) Nu and (b) f values for $\beta = 0.0, \beta = 0.21, \beta = 0.27, \beta = 0.31$ at $W/w = 6$ for perforated multi-V ribs with the smooth channel in SPSAH.

Thermohydraulic performance

The presence of “Nu” and “ f ” generates confusing scenarios in which determining the real benefits of artificial roughness applications looks difficult. Webb and Eckert [43] presented a modified Thermo-Hydraulic Performance Parameter (THPP), designated as THPP and given by equation 4, that refers heat gain to friction loss and assists in evaluating the increase in THP for proposed roughness as compared to the smooth channel for similar energy inputs and defined as:

$$THPP = \left[\frac{Nu_r}{Nu_s} \right] / \left[\frac{f_r}{f_s} \right]^{\frac{1}{3}} \quad (4)$$

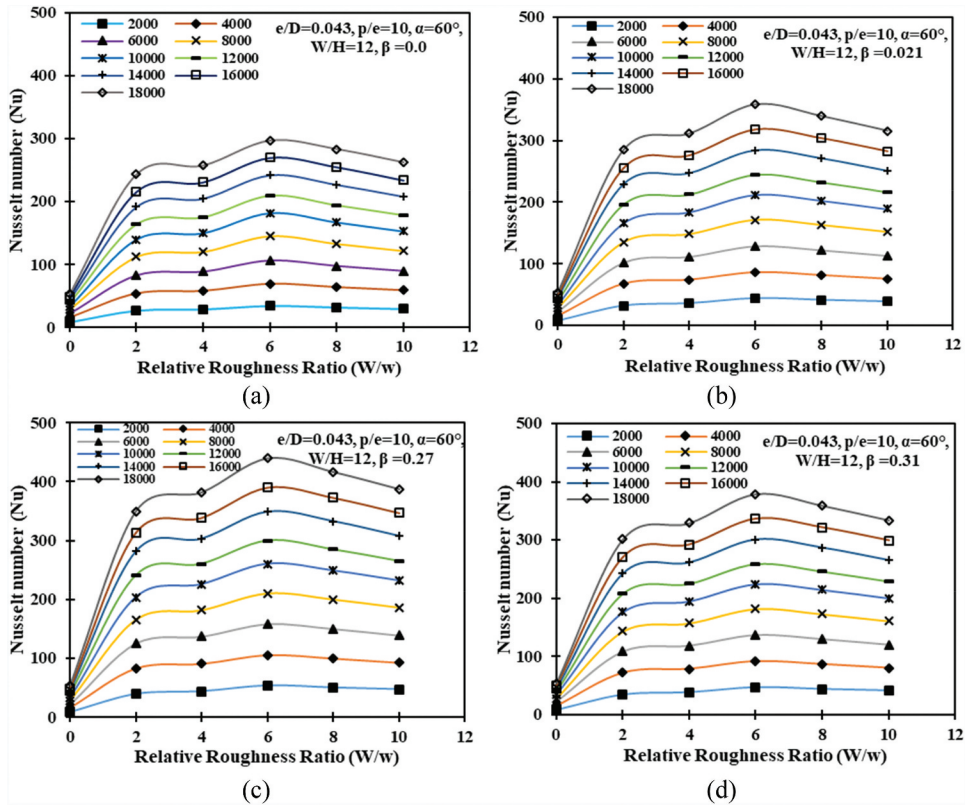


Figure 5. Influence of W/w on Nu at (a) $\beta = 0.0$, (b) $\beta = 0.21$, (c) $\beta = 0.27$, (d) $\beta = 0.31$ at $W/w = 6$ for a range of different perforated multi-V ribs in SPSAH.

A THPP value higher than one demonstrates the advantages of artificial roughness on smooth plates and may be used to find the best rib patterns for improved THP.

Figure 9(a) presents the connection between THPP and Re for various β values for perforated multi-V ribs in SPSAH, with optimum results obtained for $\beta = 0.27$. The optimum value of THPP is 11.08 for $\beta = 0.27$ at $W/w = 6$, respectively. The suggested roughness demonstrates a substantial improvement in THPP values, as compared to the smooth SAH.

Functional relationships for Nusselt number (Nu) and friction factor (f) for SPSAH

For the actual data presented in this study, the statistical correlation between “ Nu ” and “ f ” is derived through a regression analysis. The results in the previous sections show that the THPP of artificially roughened SPSAH is a strong function of design and rib geometrical factors [37] like Reynolds number (Re), open area ratio (β), and relative roughness width (W/w). As a result, the functional correlations for Nu and f are as follows:

$$Nu = (Re, \beta, W/w) \quad (5)$$

$$f = (Re, \beta, W/w) \quad (6)$$

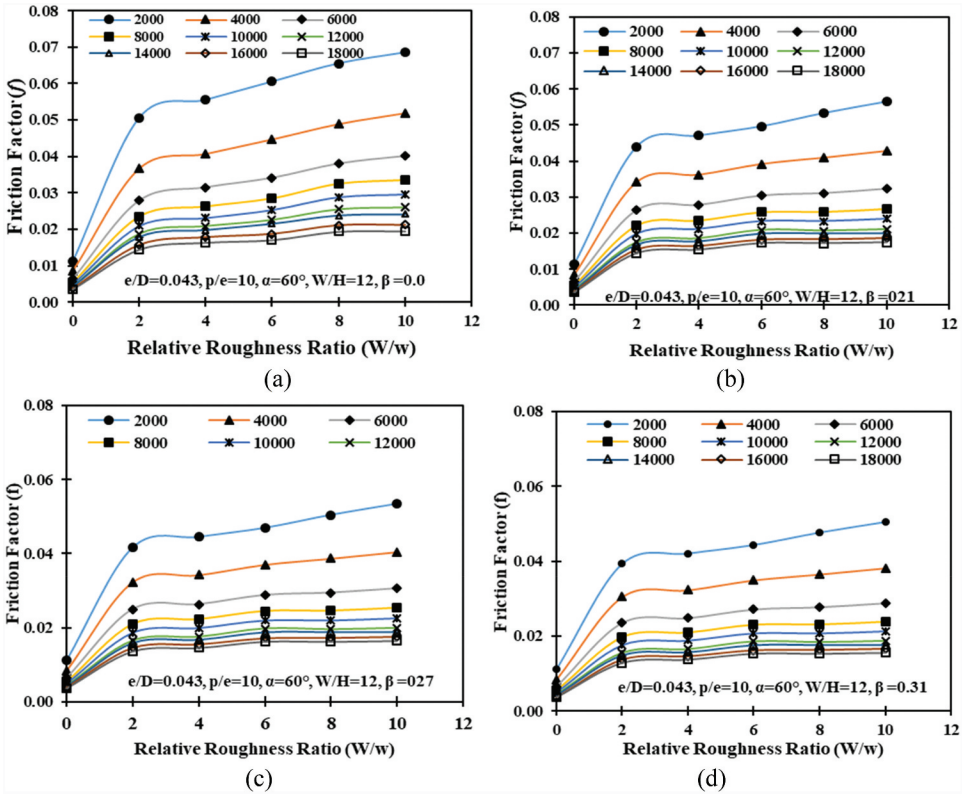


Figure 6. Influence of W/w on f at (a) $\beta = 0.0$, (b) $\beta = 0.21$, (c) $\beta = 0.27$, (d) $\beta = 0.31$ at $W/w = 6$ for a range of different perforated multi-V ribs in SPSAH.

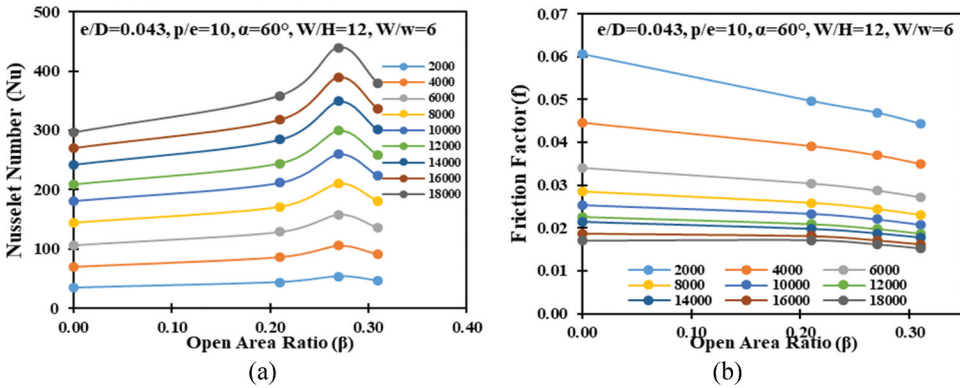


Figure 7. Influence of β on (a) Nu and (b) f values for SPSAH.

Nusselt Number (Nu) correlation

The change in the numeric value of a parameter in connection to other variables would be expressed through correlations. As a result, to obtain Nu correlation, the experimental data set was used in the following manner:

- As Re increased, Nu increased monotonically (Figure 4(a)).

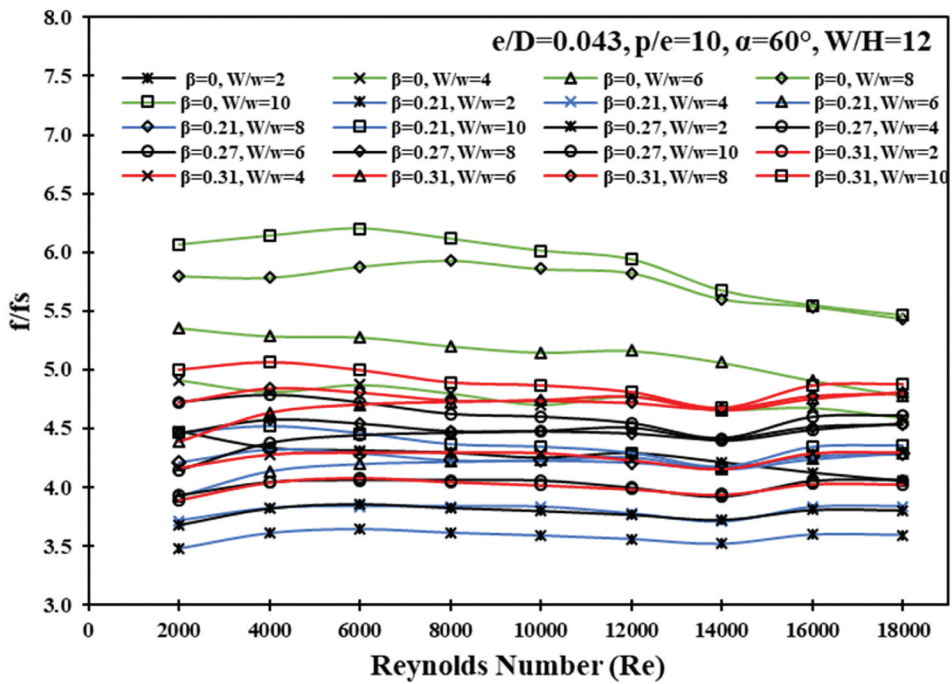
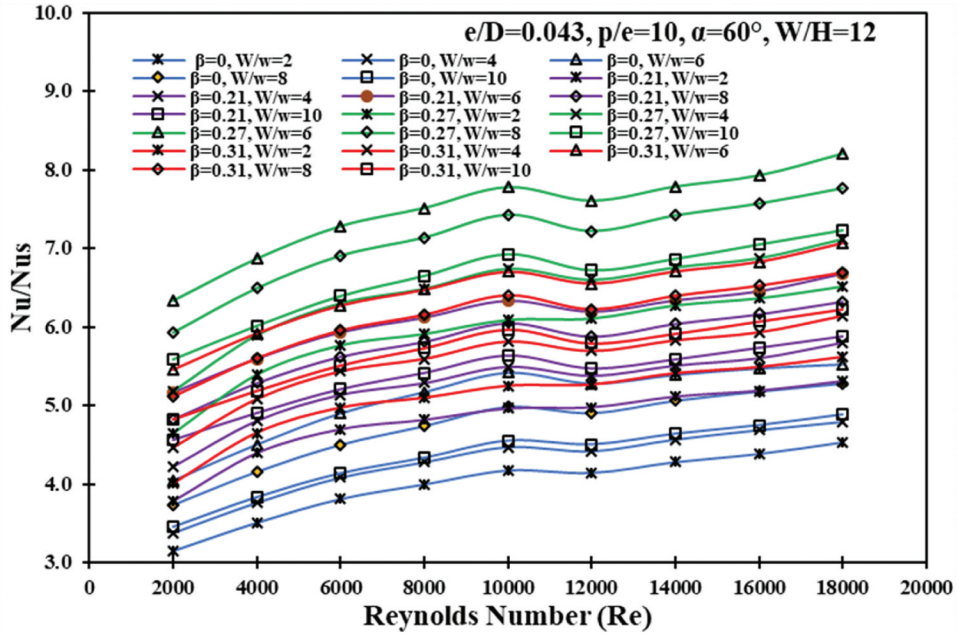


Figure 8. (a) Influence of Re on (a) Nu/Nus values and (b) f/f_s values at $\beta = 0.0, \beta = 0.21, \beta = 0.27, \beta = 0.31$ at $W/w = 6$ for perforated multi-V ribs in SPSAH

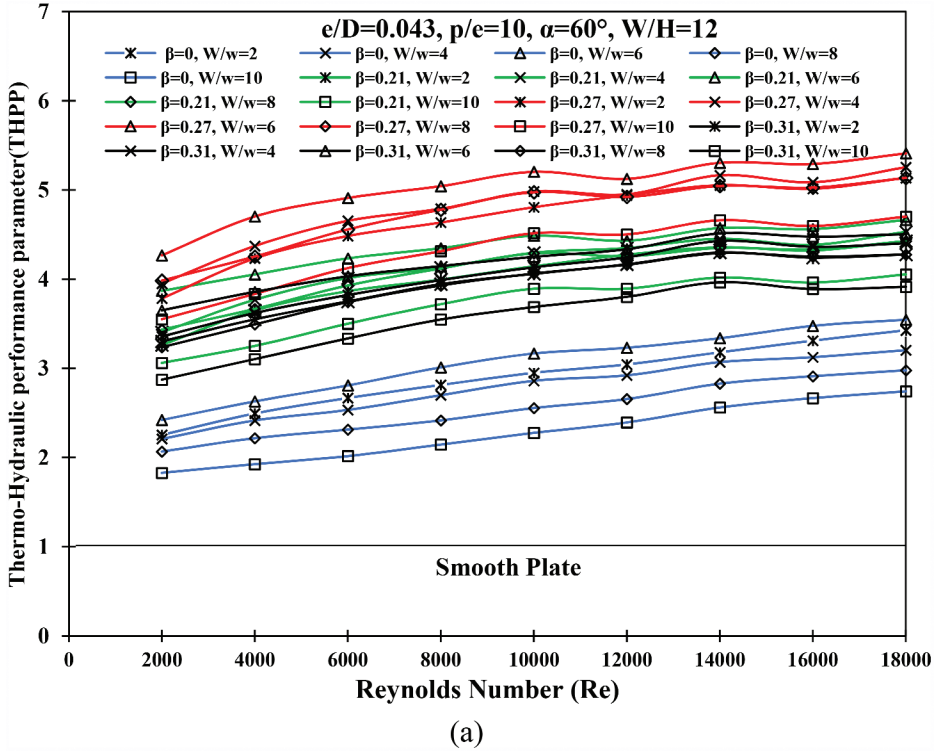


Figure 9. (a) Variation in THPP with Re for different β for perforated multi-V rib in SPSAH

- With increasing W/w value, Nu attain optimum value at $W/w = 6$ (Figure 5)
- With increasing open area ratio (β), Nu attain maximum value at $\beta = 0.27$ (Figure 7(a)).

As a result, the empirical result produces continuous smooth lines including almost uniform slopes (n) with a maximum deviation of 2.37%, but the intercept (A_0) of each line exhibits a wide fluctuation with a maximum deviation of 29.96%, as seen in Figure 10. For the remaining combinations of roughness geometric characteristics, such as W/w and β , identical graphs of the $\ln(Nu)$ vs $\ln(Re)$ have been displayed. Eq. 7 may be used to represent the trends of the line in the plot of $\ln(Nu)$ vs. $\ln(Re)$.

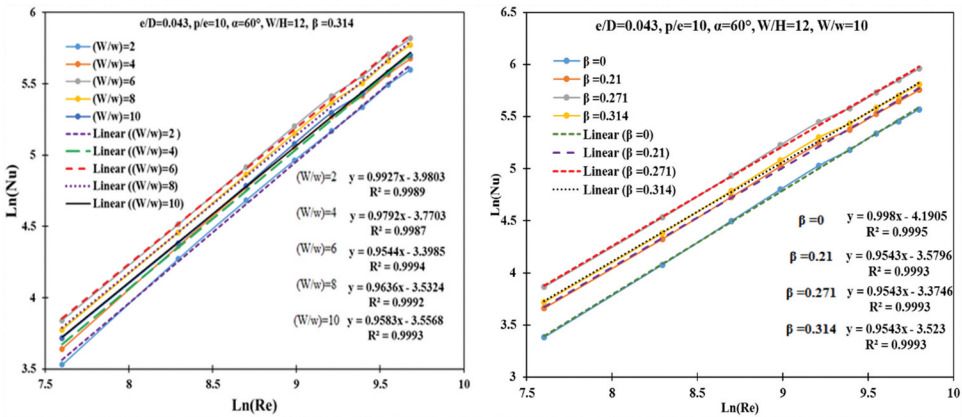


Figure 10. Graph of $\ln(Nu)$ vs $\ln(Re)$ to find out " A_0 " and slope " n " for various values of W/w and β in SPSAH.

$$\ln(Nu) = \ln(A_0) + n\ln(Re) = A_1 + n\ln(Re) \quad (7)$$

Eq.7 can also be written as:

$$Nu = A_0 Re^n \quad (8)$$

Where $A_0 = \text{Anti Ln}(A_1)$

As illustrated in [Figure 11](#), the least square technique is used to define the appropriate curve throughout the whole range of statistical data sets connected to 21 distinct sets of test runs performed, and the resulting association can be expressed as:

$$Nu = A_0 Re^{0.9648} \quad (9)$$

The value of coefficient A_0 in Eq. 9 for SPSAH is determined by other influencing characteristics such as β and W/w . For SPSAH, to carry out the influence of β on Nu , the value of $A_0 = Nu/(Re)^{0.9648}$ corresponding to all values of W/w and for some specific values of Re has been plotted against relative perforated rib as illustrated in [Figure 12](#). The second-order quadratic equation describes the best fit curve for all the data points. Therefore, regression analysis to fit a second-order polynomial relation on a log-log graph through the data sets is shown in [Figure 12](#), which yields:

$$\ln\left[\frac{Nu}{Re^{0.9648}}\right] = \ln(B_1) + B_2 \ln(\beta) + B_3 [\ln(\beta)]^2 \quad (10)$$

It can be rearranged as

$$\frac{Nu}{Re^{0.9648}} = B_0(\beta)^{B_2} \text{Exp}[B_3 (\ln(\beta))^2] \quad (11)$$

Where $B_0 = \text{Anti Ln}(B_1)$

Substituting the values of B_2, B_3 from the curve fitting results in the following expressions:

$$\frac{Nu}{Re^{0.9648}} = B_0(\beta)^{0.2338} \text{Exp}[1E^{-12}(\ln(\beta))^2] \quad (12)$$

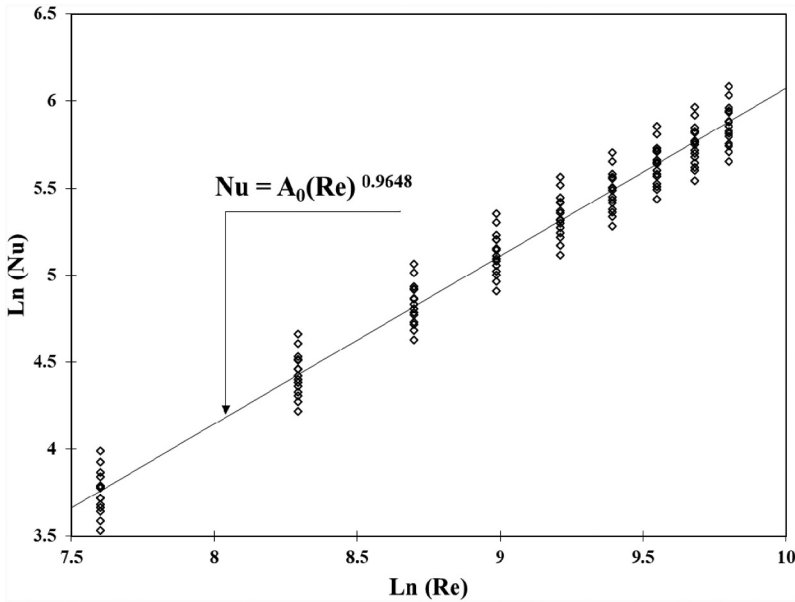


Figure 11. $\ln(Nu)$ vs $\ln(Re)$ for the entire range of data sets for SPSAH.

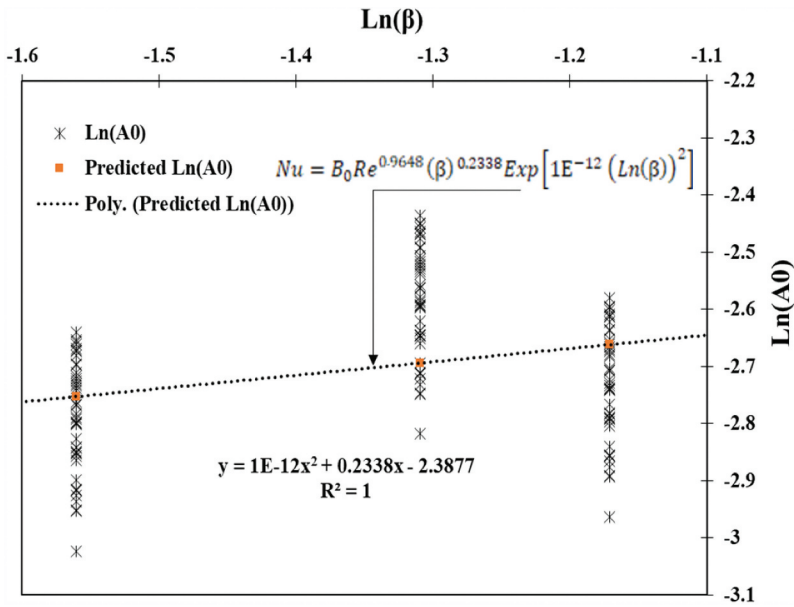


Figure 12. Graph of $\text{Ln}(A_0)$ vs. $\text{Ln}(\beta)$ for SPSAH.

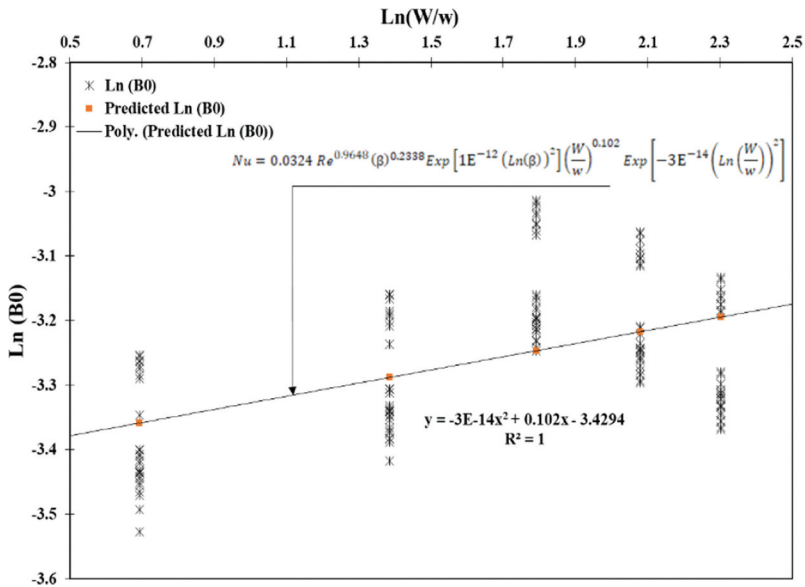


Figure 13. $\text{Ln}(B_0)$ vs. $\text{Ln}(W/w)$ for SPSAH.

Different contributing factors influence the coefficient B_0 in Eq.12, namely W/w , and graphing $\text{Ln}(Nu)$ vs $\text{Ln}(W/w)$ for SPSAH produces the required results. To obtain the functional relationship between $\text{Ln}(B_0)$ and $\text{Ln}(W/w)$, Figure 13 has been drawn, which shows the second-order polynomial expression to describe the best fit curve. Therefore, considering the parameters W/w the values of $B_0 = \frac{Nu}{Re^{0.9648} (\beta)^{0.2338} \text{Exp} [1E^{-12} (\text{Ln}(\beta))^2]}$ have been plotted against W/w in Figure 13. The following second-order polynomial relation has been obtained to best fit the data.

$$\text{Ln} \left[\frac{\text{Nu}}{\text{Re}^{0.9648} (\beta)^{0.2338} \text{Exp} [1\text{E}^{-12} (\text{Ln}(\beta))^2]} \right] = \text{Ln}(C_1) + C_2 \text{Ln} \left(\frac{W}{w} \right) + C_3 \left(\text{Ln} \left(\frac{W}{w} \right) \right)^2 \quad (13)$$

This can further be reduced to,

$$\frac{\text{Nu}}{\text{Re}^{0.9648} (\beta)^{0.2338} \text{Exp} [1\text{E}^{-12} (\text{Ln}(\beta))^2]} = C_0 \left(\frac{W}{w} \right)^{C_2} \text{Exp} C_3 \left(\text{Ln} \left(\frac{W}{w} \right) \right)^2 \quad (14)$$

Where $C_0 = \text{Anti Ln}(C_1)$

Substituting the values of C_2, C_3 obtained from the regression of the curve fitting gives,

$$\frac{\text{Nu}}{\text{Re}^{0.9648} (\beta)^{0.2338} \text{Exp} [1\text{E}^{-12} (\text{Ln}(\beta))^2]} = C_0 \left(\frac{W}{w} \right)^{0.102} \text{Exp} \left[-3\text{E}^{-14} \left(\text{Ln} \left(\frac{W}{w} \right) \right)^2 \right] \quad (15)$$

or

$$\text{Nu} = C_0 \text{Re}^{0.9648} (\beta)^{0.2338} \text{Exp} [1\text{E}^{-12} (\text{Ln}(\beta))^2] \left(\frac{W}{w} \right)^{0.102} \text{Exp} \left[-3\text{E}^{-14} \left(\text{Ln} \left(\frac{W}{w} \right) \right)^2 \right] \quad (16)$$

The magnitudes of coefficients $A_0 = 0.02796$, $B_0 = 0.0918$ and $C_0 = 0.0324$ for SPSAH are obtained from fitting the curves in [Figure 11 to 13](#).

The final correlation for Nu can be expressed as:

$$\text{Nu} = 0.0324 \text{Re}^{0.9648} (\beta)^{0.2338} \text{Exp} [1\text{E}^{-12} (\text{Ln}(\beta))^2] \left(\frac{W}{w} \right)^{0.102} \text{Exp} \left[-3\text{E}^{-14} \left(\text{Ln} \left(\frac{W}{w} \right) \right)^2 \right] \quad (17)$$

Friction Factor (f) correlation

The same approach was used to calculate the correlation for f as it was for Nu. Friction factor (f), like Nu, was shown to be closely linked to roughness design variables (W/w and β) as well as flow variables (Re). The following is the effect on f values as a function of different roughness variables:

- As Re increased, f decreases monotonically ([Figure 4\(b\)](#)).
- With increasing W/w , f increases and attains a optimum value at $W/w = 10$ ([Figure 6](#)).
- With increasing open area ratio (β), f decreases and attend minimum value at $\beta = 0.31$ ([Figure 7 \(b\)](#)).

Data from the experiments was utilized to do regression analysis, which provided the best fit curve, through which the intercept and slope coefficients for the f correlation were calculated.

The change in f as a function of Re is seen in [Figure 14](#). With empirically collected data, regression analysis was done to obtain the following best-fit curve:

$$\text{Ln}(f) = \text{Ln}(D_0) + n\text{Ln}(Re) = D_1 + n\text{Ln}(Re) \quad (18)$$

$$f = D_0(Re)^{-0.529} \quad (19)$$

Where $D_0 = \text{Anti Ln}(D_1)$

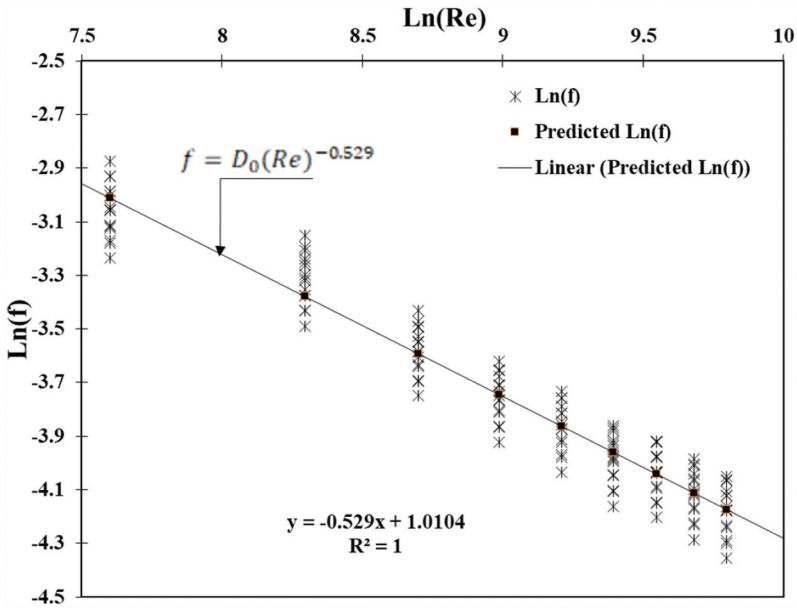


Figure 14. Graph of $\text{Ln}(f)$ as a function of $\text{Ln}(Re)$ for all empirical data sets of SPSAH.

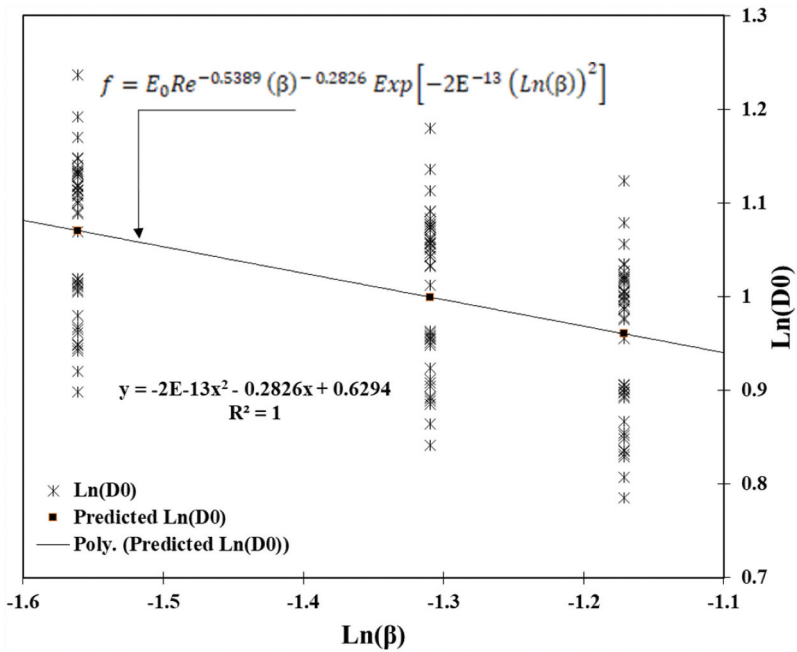


Figure 15. Graph of $\text{Ln}(D_0)$ as a function of $\text{Ln}(\beta)$ in case of SPSAH.

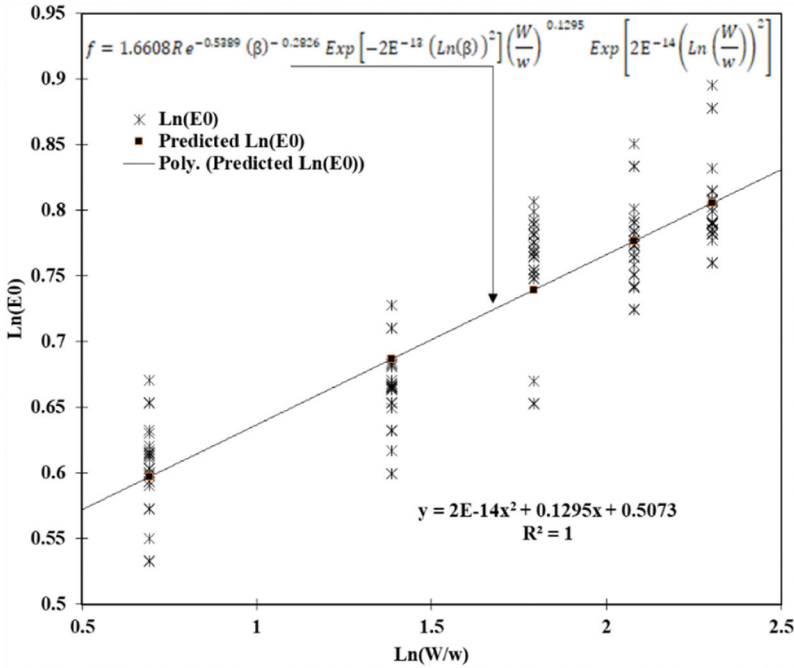


Figure 16. Graph of $\text{Ln}(E_0)$ as a function of $\text{Ln}(W/w)$ in case of SPSAH.

For SPSAH, the coefficient D_0 is as a result of other contributing factors as, β , and W/w . To bring out the effect of the values of $D_0 = \frac{f}{Re^{-0.529}}$ corresponding to all values of open area ratio (β) are plotted against perforated multi-V rib as shown in Figure 15, that shows a regression technique to draw a second-order polynomial curve on a log-log graph across the data sets. can be express as:

$$\text{Ln} \left[\frac{f}{Re^{-0.529}} \right] = \text{Ln}(E_1) + E_2 \text{Ln}(\beta) + E_3 [\text{Ln}(\beta)]^2 \quad (20)$$

It can be rearranged as,

$$\frac{f}{Re^{-0.529}} = E_0(\beta)^{E_2} \text{Exp}[E_3 (\text{Ln}(\beta))^2] \quad (21)$$

Where $E_0 = \text{Anti Ln}(E_1)$

Substituting the values of E_2 , E_3 from the curve fitting results in the following expressions;

$$\frac{f}{Re^{-0.529}} = E_0(\beta)^{E_2} \text{Exp}[E_3 (\text{Ln}(\beta))^2] \quad (22)$$

Or

$$\frac{f}{Re^{-0.529}} = E_0(\beta)^{-0.2826} \text{Exp}[-2E^{-13} (\text{Ln}(\beta))^2] \quad (23)$$

The coefficient E_0 in Eq. 23 is a result of other contributing variables, namely W/w , and the desired plots can be obtained by plotting $\text{Ln}(f)$ vs $\text{Ln}(W/w)$ for SPSAH. To obtain the functional relationship between $\text{Ln}(E_0)$ and $\text{Ln}(W/w)$, Figure 16 has been drawn, which shows the second-order polynomial expression to describe the best fit curve. Therefore, considering the parameters W/w the values of $E_0 = \frac{f}{Re^{-0.529} (\beta)^{-0.2826} \text{Exp}[-2E^{-13} (\text{Ln}(\beta))^2]}$ have been plotted against W/w in Figure 16.

The following second-order polynomial relation has been obtained to best fit the data.

$$\text{Ln} \left[\frac{f}{\text{Re}^{-0.529} (\beta)^{-0.2826} \text{Exp}[-2\text{E}^{-13} (\text{Ln}(\beta))^2]} \right] = \text{Ln}(F_1) + F_2 \text{Ln} \left(\frac{W}{w} \right) + F_3 \left(\text{Ln} \left(\frac{W}{w} \right) \right)^2 \quad (24)$$

This can further be reduced to,

$$\frac{f}{\text{Re}^{-0.529} (\beta)^{-0.2826} \text{Exp}[-2\text{E}^{-13} (\text{Ln}(\beta))^2]} = F_0 \left(\frac{W}{w} \right)^{F_2} \text{Exp} \left[F_3 \left(\text{Ln} \left(\frac{W}{w} \right) \right)^2 \right] \quad (25)$$

Where $F_0 = \text{Anti Ln} (F_1)$

Substituting the values of F_2 and F_3 obtained from the regression of the curve fitting gives,

$$\frac{f}{\text{Re}^{-0.529} (\beta)^{-0.2826} \text{Exp}[-2\text{E}^{-13} (\text{Ln}(\beta))^2]} = F_0 \left(\frac{W}{w} \right)^{0.1295} \text{Exp} \left[-9\text{E} - 15 \left(\text{Ln} \left(\frac{W}{w} \right) \right)^2 \right] \quad (26)$$

$$f = F_0 \text{Re}^{-0.529} (\beta)^{-0.2826} \text{Exp}[-2\text{E}^{-13} (\text{Ln}(\beta))^2] \left(\frac{W}{w} \right)^{0.1295} \text{Exp} \left[2\text{E}^{-14} \left(\text{Ln} \left(\frac{W}{w} \right) \right)^2 \right] \quad (27)$$

The values of coefficients $D_0 = 2.7469$, $E_0 = 1.876$ and $F_0 = 1.6608$ for SPSAH are obtained from fitting the curves in [Figure 14 to 16](#).

The final correlation for f can be expressed as:

$$f = 1.6608 \text{Re}^{-0.529} (\beta)^{-0.2826} \text{Exp}[-2\text{E}^{-13} (\text{Ln}(\beta))^2] \left(\frac{W}{w} \right)^{0.1295} \text{Exp} \left[2\text{E}^{-14} \left(\text{Ln} \left(\frac{W}{w} \right) \right)^2 \right] \quad (28)$$

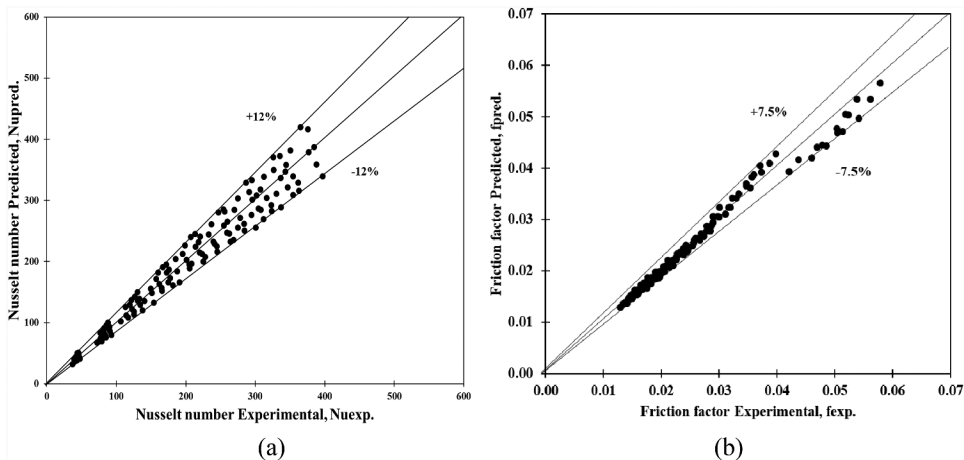


Figure 17. Comparison of empirical and projected results of (a) Nu (b) f for developed correlation for SPSAH.

Deviation in theoretical and experimental Nusselt number (Nu) and friction factor (f)

Figure 17(a,b) compares empirical Nu and f data with predicted Nu and f projections and signifies that approximately 98% of the data points are located within $\pm 12\%$ and $\pm 7.5\%$ deviation lines with regression coefficient ($R[2]$) of the correlations are 0.95 and 0.96, respectively. As a result, the established correlation equations (17) and (28) may be implemented to forecast the Nu and f within acceptable ranges for the parameters studied in this study.

Conclusions

The THP and correlation studies in a SPSAH duct with different perforated multi-V ribs exposed to an equally distributed thermal gradient of radiant heat demonstrate that perforated multi-V ribs significantly boost THP. The following are the findings obtained from the aims of the present study:

- The optimum performance of perforated multi-V rib roughness was found with an open area ratio (β) of 0.27 in all cases.
- The value of Nu attained its optimal value at $W/w = 6$, while the f value increases as W/w increases.
- The highest improvement in Nu/Nus and f/fs as compared to smooth channels was found 8.19 times and 4.78 times, respectively.
- The optimum value of THPP was observed as 5.41 for perforated multi-V rib roughness with an open area ratio (β) of 0.27.
- Within the pre-defined range of parameters W/w , β and Re , correlations for Nu and f developed with $\pm 12\%$ and $\pm 7.5\%$ of forecasting, respectively.

As shown in this study, the optimal value of the open area ratio for perforated multi-V ribs improves thermos-hydraulic performance significantly when compared to a smooth duct. Hence, the improved findings indicate the possibility of using perforation in various ribs in artificially roughened SAH. Further study can also be applied to double-pass solar air heaters with various rib roughness.



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Disclosure statement

No potential conflict of interest was reported by the authors.

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Nomenclature

Details of symbols	Dimensionless parameters
D_h Duct hydraulic diameter, (m)	e/D_h Relative roughness height
e Elevation of rib, (m)	f duct friction
H Elevation of channel/duct, (m)	Nu Roughened plate Nusselt number

(Continued)

Details of symbols	Dimensionless parameters
\dot{m} Air mass-flow rate, (kg/s)	P/e Relative roughness pitch
W Width of duct channel, (m)	Re Reynolds number
w Width of one set of rib, (m)	W/H Channel aspect ratio
Abbreviations	W/w Relative roughness width
CFD Computational Fluid Dynamics	Greek symbols
THP Thermo-Hydraulic Performance	α Angle of attack, (°)
SPSAH Single Pass Solar Air Heater	β Collector slope, the open area ratio

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